THE SIMULATION MODEL OF AUTOMOBILE COLLISIONS (SMAC) OPERATOR'S MANUAL

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SECTION II

AN OVERVIEW OF OPERATING THE SMAC PROGRAM

The SMAC program user specifies values of input variables describing the vehicles, the accident scene, and some computation details.* The computer will return a printout giving position, heading, vector and angular velocity, as well as tire tracks, at regular time intervals throught—out the collision sequence. If desired, a graphic display of the simulated collision will also be created by a plotting subroutine. If the user is trying to simulate an actual collision, he will note differences between the actual and simulated collision, and modify the input data to obtain a better match.**

To clarify the data input procedure, we shall divide input data into four categories: vehicle properties (length, width, mass, etc.). calculation constants, initial conditions, and control inputs. Where particular vehicles are being simulated, most properties for any given make and model of car car be found in a reference manual (see reference 1). Otherwise, typical values such as those provided in the attached Table I may be used. Usually, values for vehicle properties are not changed in the iteration process mentioned above.

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^{*} We suggest using an input data form provided with the manual to record your decisions prior to transferring them on the computer (See Section III(D)).

^{**} The art of modification is a subject in itself. One suggestion is to modify one variable at a time in order to attribute the total change from the last run to the present run to that sole modification. Understanding the theory of the SMAC program and understanding how a field investigator chooses a value for a variable will give you much insight into the modification phase of SMAC.

Calculation constants, such as the time interval used, the acceptable error in balancing collision forces, etc., are explained individually in this manual, and suggested values are given. Generally, changes in these variables will only have a secondary effect on accuracy of calculation, expense of run, or specific error messages discussed subsequently.

Often, the user will wish to simulate an accident in which only part of the information, say the final resting positions and tire tracks, is known. He will vary the input data, trying to obtain outputs corresponding to collision. The input variables of interest are the initial conditions (position, heading, velocity) and the control inputs (braking, accelerative traction, and steering). Once the skid marks and final positions match fairly well, the user will presumably have obtained a good approximation of the initial positions, headings and velocities of the cars, if the vehicle properties used are accurate.

SECTION III

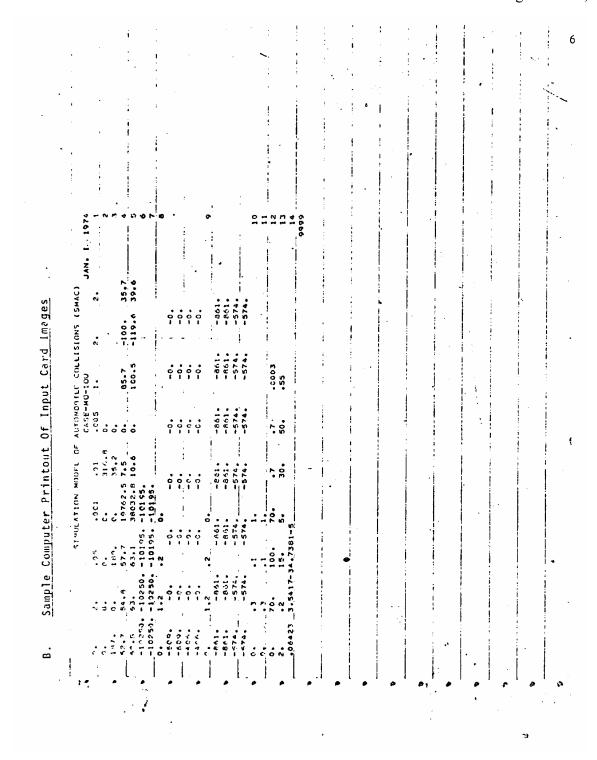
INPUT FORMAT*

A. Discussion

The first two records (e.g. cards) of a data deck for input to the SMAC program (if a data file is used, one line = one record) are heading records: these are for user convenience and contain no calculation data. Information placed in these records is printed at the top of each page of output to identify the printout. While they may be left blank, these records must not be omitted.

Next come 14 numbered records giving calculation input data. (In certain cases discussed later there will be additional, unnumbered records following records 8 and 11. These are the torque and steering tables.) The numbered records 1 through 14 are formatted 9F8.0, I8. (A remote time-share terminal user has the capability to input the input records in a free format.) Calculation data appear in the 9 floating point fields. A decimal point must appear in each floating point number. The card identification number appears, right justified, in the final integer field. After record 14 a final record, blank except for the number 9999 in columns 77-80, completes the input data. A sample computer printout of the input card images of a run follows in Section III B.

^{*}Section III is subdivided into four parts which should be studied together. Once SMAC format is absorbed, the user is ready to learn the definitions of the input variables (i.e., Section IV).



1

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SECTION IV DESCRIPTION OF INPUT VARIABLES*

A. Detailed Input Variable Description

Input data variables are listed below in the following manner:

- (1) A pointer denotes the card and the field position within that card (e.g., the pointer 1:3 refers to card 1, field 3).
- (2) The variable name as it occurs in the program (e.g., DTTRAJ).
- (3) Where applicable, a symbol is used in analysis, usually a Greek letter with appropriate subscript.
- (4) An explanation of the variable is given.
- (5) Where called for, suggested values are given.

(Note 0 indicates zero, 0 indicates letter "oh").

- 1:1 TO The time, in seconds, at which the program starts. TO is arbitrary, and is generally chosen to be 0.0 for convenience.
- 1:2 TF The time, in seconds, at which the program ends.
 TF-T0=total duration of the run.
- 1:3 DTTRAJ The program approximates a continous, non-linear path in time and space by breaking it into small increments, taken as linear, rather than by a true line integral. DTTRAJ is the time interval of integration before, and again after, vehicle contact. This will generally be the largest time interval used (on the order of .05 seconds, smaller where forces or speeds are usually high), since changes are gradual and continous during trajectory.

[With this, and with all intervals, a smaller value will yield greater accuracy, but will also increase computer time and expense. In general, the accuracy desired will be greater when the user wishes to match a set of empirical data than when seeking to simulate a situation for heuristic purposes.]

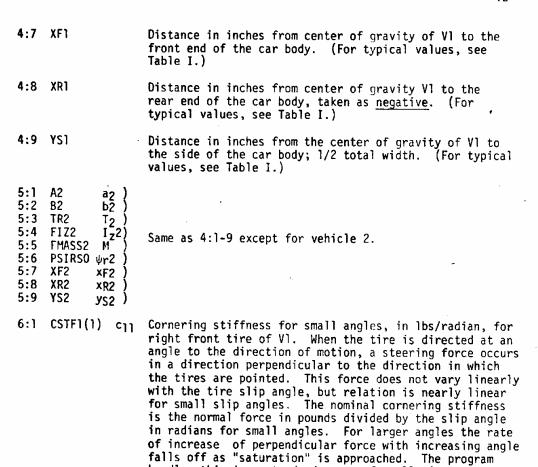
1:4 DTCOLL This is the interval of integration, in seconds, during the collision, where large crash forces require a small time interval (on the order of .001) to obtain a good approximation.

^{*}It is recommended that until becoming "adequate" in the utilization of SMAC that the user rely on the detailed input variable description of Section IV(A) rather than the brief input variable description of Section IV(B).

1:5 DTCOLT		This is the interval of integration in seconds for the first 100 time increments immediately after vehicle separation. DTCQLT will generally be smaller than DTTRAJ (e.g., .01) since higher speeds and spin are often involved at this stage.
1:6 DTPRNO		The printout time interval in seconds.
1:7 UVMIN 1:8 PSIDMN)	If the absolute value of the total vector velocity in inches/second is less than UVMIN for both vehicles, and the absolute value of the angular velocity in degrees/second is less than PSIDMN for both vehicles, the run terminates. If no minimum value cut-off is desired, these can be set to 0.0. (1 mph = 17.6 inches/second)
1:9 IVEHO		Number of simulated vehicles (1. or 2.) if IVEH0 = 1., program ignores inputs on records 3, 5,7,9 & 11; however, these records must be included in keeping with format.
1:10		The numerals 01, 02, $-$, 14 go in columns 79 and 80 (see Section III(C)). This item is omitted on subsequent record explanation.
2:1 XCP10	Х'с10	The X' coordinate, in inches, of the center of gravity of of vehicle 1, (VI). The smaller vehicle should be entered as VI Collision forces are calculated in a clockwise sweep of VI. The force calculations are less accurate in the vicinity of a narrow intrusion; therefore, accuracy is improved when VI is the smaller vehicle. A fixed Cartesian coordinate system is used, with the positive X' axis shown pointing upward, and the positive Y' axis to the right. Angles are measured clockwise from the positive X' axis.
2:2 YCP10	Y'c10	The Y'-coordinate, in inches, of the center of mass of V1. (See also comments on XCP10).
2:3 PSI10	Ψ1 0	The heading angle, in degrees of V1 measured clockwise from the positive X' axis.
2:4 PSI1DO) ^ψ 10	The angular velocity, in degrees/second of V1. Taken to be positive when rotation is clockwise.
2:5 U10	Ψ10	The initial forward velocity of VI in inches/second. The longitudinal component of the total vector velocity of VI.

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2:6	v 10		The initial sideway velocity of V1 in inches/second with right taken as positive.
3:1	XCP20 YCP20 PSI20 PSI20 U20 V20	X'C20 Y'C20 Ψ20 Ψ20 Ψ20 Ψ20 Ψ20	Same as Card 2; except for vehicle 2.
4:1	Al	al .	The distance in inches from the center of gravity of V1 to the midpoint between the front wheels (see Table I for typical values).
4:2	В1	aj	The distance in inches from the center of gravity of VI to the midpoint between the rear wheels, taken as positive (see Table I for typical values).
4:3	TR1	т	Average tread width in inches, i.e., distance between left and right tires, averaged over front and rear pairs (for typical values, see Table I).
4:4	FIZI	I _{z1}	Yaw inertia, in $1b\text{-sec}^2$ -inches, of V1. This is a measure of the torque needed to induce a given spin in V1, and depends both on the total mass of the vehicle and on how far this mass is, on the average, from the center of gravity. For typical values of mass and of k^2 ($I_ZI = k^2 \times M$), see Table I.
4:5	FMASS1	М	The total mass of VI, measured in lb-sec ² /inch. If vehicle weight in lbs. is known, mass in lb-sec ² /inch can be found by dividing by 386.4 (for typical values see Table I).
4:6	PSIR10	Ψ _R 1	The rear axle steer angle in degrees; angular displacement from normal orientation, with clockwise displacement taken as positive (thus for undamaged rear axle = 0.0).



Typical values for cornering stiffness are -10200 lbs/radian. Cornering stiffnesses are input separately to allow the simulation of tire damage--a damaged tire which has lost its pressure will have a far lower cornering stiffness, perhaps 20 percent of that for an undamaged tire. Under-inflated tires will have somewhat lower cornering stiffnesses than the values given, but far greater than seriously damaged tires.

handles this in a standard manner for all tires.

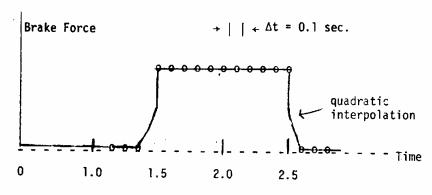
CSTF1(2)	c ₁₂	Cornering stiffness, left front tire of V1, 1bs/radian.
CSTF1(3)	c ₁₃	Cornering stiffness, right rear tire of V1, lbs/radian.
CSTF1(4)	C ₁₄	Cornering stiffness, left rear tire of V1, lbs/radian.
CSTF2(1)	c ₂₁₎	
CSTF2(2)	c ₂₂ }	,
CSTF2(3)	C ₂₃	Same as 6:1-4, except for V2.
CSTF2(4)	c ₂₄ }	
твтол		Initial time in seconds for V1's Torque (braking or accelerating) inputs. When applying torque to V1 after the run has begun (TBTQ1>TO), at least three zero inputs prior to time TBTQ1 should be inputed. This is done to accommodate quadratic interpolations which require two end points and one turning point (thus 3 points).
ТЕТОІ		End time for torque inputs for VI in seconds. If control inputs for torque end in the middle of a run (TETQI <tf) (same="" 8:1).<="" after="" as="" control="" end="" input="" inputs="" insure="" reasoning="" should="" tables="" td="" tetqi="" three="" to="" with="" zero=""></tf)>
TINCQ1		Time increment for torque inputs, VI, in seconds. One value for torque is input each increment, so the size of TINCQ determines the fineness with which braking and traction can be described by the mathematical model.
NTBLQ1		If this variable is set equal to zero, the program reads the torque input tables and includes them in the calculations for the run. If any other value is entered the program ignores card #8 and there are no torque inputs for VI. For cards #8-11, when the final variable is set equal to 0.0, the card is followed by tables, each consisting of from 1 to 29 unnumbered cards, formatted (7F10.0). Each table represents the control inputs for a single wheel. Accelerative torque inputs are positive, braking inputs negative, both are in pounds. The number of entries per wheel is TETQ1 - TBTQ1 + 1. Entries for each wheel begin on TINCQ1 a new card.
	CSTF1(3) CSTF1(4) CSTF2(1) CSTF2(2) CSTF2(3) CSTF2(4) TBT01 TETQ1	CSTF1(3) C ₁₃ CSTF1(4) C ₁₄ CSTF2(1) C ₂₁ CSTF2(2) C ₂₂ CSTF2(3) C ₂₃ CSTF2(4) C ₂₄ TBTQ1 TETQ1

8:4 (Continued)

If a friction decrement with speed is used (item 12:7), at high speeds, friction and thus maximum possible tire force will be reduced. Also, if both steering and traction inputs are included for a given tire, the vector sum of these inputs cannot exceed the maximum possible force for that tire. If too large a force is entered, the program will substitute the largest possible value (i.e., a value equal to the product of the weight on that tire multiplied by the coefficient of friction).

To illustrate the torque input process, an example is in order.

Example of Torque Input. It is desired to apply the brakes of the rear wheels of a 4,000-lb vehicle (VI) strong enough to lock those wheels during the interval from 1.5 to 2.5 seconds after the program starts. Brake forces are zero at other times. (Steering inputs would be handled in the same manner as this braking example.)



Card 8 1.2 2.8 0.1 0.

(Plus 3 cards for LF wheel identical to ...

8:4 (Continued)

Notice that the start and end times are three time increments before and after the brake pulse, respectively, to allow for insertion of three zero force inputs as discussed under 8:1 and 8:2.

Notice that the braking force input (-2000 lbs. on each rear wheel) exceeds the maximum friction force available at that wheel (approximately 1/4 of vehicle weight x friction coefficient). This is done to insure skidding. The program will combine this force vectorially with any steering forces present, and limit the resultant to the maximum friction force.

Tables for Card 8:

- (1) Traction (+) or braking (-) inputs, right front wheel, VI, in pounds force.
- (2) Same, left front wheel.
- (3) Same, right rear wheel.
- (4) Same, left rear wheel.

9:1 TBTQ2 9:2 TETQ2 9:3 TINCQ2 9:4 NTBLQ2 Tables (1-4)	Same as 8:1-4 and subsequent tables, except for V2.
10:1 TBPSF1	Starting time for steer inputs, in seconds, V1.' (See 8:1; same format as torque inputs.)
10:2 TEPSF1	End time for steer inputs, in seconds, V1 (see 8:2).
10:3 TINCP1	Time increment for steer inputs, in seconds, VI (see 8:3).
10:4 NTBLP1	If ≠0.0, program ignores card 10 and no steering inputs are used in calculations for V1.
	Steer tables are exactly like torque tables (see note on control inputs after card 8) except that values input represent tire angle, in degrees (left turn is negative, right is positive, straight ahead is 0.0 degrees). There are only 2 tables per vehicle since only front wheels are steered.
	Tables for Card 10:
	(1) Steer inputs, right front tire, VI, in degrees.
	(2) Steer inputs, left front tire, VI, in degrees.
11:1 TBPSF2)	
11:2 TEPSF2	
11:3 TINCP2	Same as 10:1-4 and subsequent tables, but for V2.
11:4 NTBLP2) Tables (1-2))	

12:1 12:2 12:3 12:4	XBP(1) YBP(1) XBP(2) YBP(2)	x'B1 y'B1 x'B2 y'B2	These coordinates, in inches, define two points, $B_1 = (x'B_1, y'B_1)$ and $B_2 = (x'B_2, yB_2)$ in the plane of the tire forces. These 2 points determine a boundary line, dividing the plane into two zones which may be assigned different coefficients of friction. This allows the user to simulate a vehicle running onto a dirt shoulder field, etc., or any such situation which demands two adjacent areas of different frictional properties. The zone on the side of the line which contains the origin is defined as zone 1, and the other zone 2.
12:5	TUMX .	μj	The coefficient of friction in zone 1. The coefficient can be made to vary with speed (see 12:7); it is assumed independent of other factors within a zone. Use 0.7 for dry pavement, 0.3 for wet.
12:6	XMU2	^μ 2	The coefficient of friction in zone 2 (see above).
12:7	CMU	с _µ	The coefficient of linear decrement of friction with tire speed in seconds/inch. In general, the effective tire-surface friction coefficient decreases with speed and c_μ simulates this decrease. The effective friction coefficient, used in all SMAC calculations, is computed as μ effective = μ_0 - $c_\mu \nu $ where μ_0 is the nominal coefficient of friction, and is the tire speed. Thus, if no decrement is desired as speed increases, is set to 0.0. The recommended value is .0003.
13:1	DELPSO	Δψ	The interval between radial vectors in degrees. The SMAC program calculates all collision forces in a clockwise sweep about the center of gravity of VI. The sweep is broken into increments of size (360 $\Delta\psi$ DELPSO must be an integar). Too large a value of $\Delta\psi$ will cause inaccuracies, too small a value will exceed the program's capacity. The collision interface is handled in a table of up to 100 points. If more than 100 points are required to handle the damage area, the message "ISTOP = 9" appears on the printout. The suggested value is 2. degrees, which can be increased (e.g. to 3.) if the error message "ISTOP = 9" appears.
13:2	DELR#0	Δρ	The increment of change of the radius vector in inches. (See explanation after card 13.) A value of around .2 is recommended.

13:3	ALAMB	λ	The acceptable error in intervehicle pressure equilibrium, in pounds/inch. Choose λ greater than Kv1 times $\Delta\rho$ and also greater than Kv2 times $\Delta\rho$ (see explanation after card 13) e.g. for Kv1 = 30., Kv2 = 50., $\Delta\rho$ = .2, λ could be 12.
13:4	ZETAV	ζ γ	The minimum relative velocity for intervehicle friction, in inches/second. If, while in contact, the adjacent surfaces of the two vehicle move with respect to each other at a speed less than this, intervehicle friction is ignored. A value of around 5.0 inch/second is recommended.
13:5	AKV(1)	K _V }	Load-deflection characteristic, in pounds/inch ² , of VI (see explanation after card 13). Values range from around 30. for subcompact cars to around 50. for full-sized.
13:6	AKV(2)	K _{v2}	Same as above, but for V2.
13:7	AMU	μ	This is the coefficient of friction between the two vehicles when they are in contact and the adjacent surfaces of V1 and V2 are moving with respect to each other (at a velocity greater than ζv). The force tangent to the interface opposing the relative motion = ζ times the force with which the surfaces press together. A value of about .55 is recommended.

Note on $\Delta \rho$, λ , K_V] and calculation of collision forces:

As the body of a car is crushed, it exerts a force proportional to that crush. The SMAC program assumes that at any point on the interface between two vehicles, the pressures exerted by the two surfaces must be essentially equal. Since the simulation is two-dimensional, units for pressure are in pounds/inch. Since the pressure is (assumed) proportional to the depth of crush, with the car body assumed to be homogeneous, the load-deflection characteristic K_V is in units of pressure/inch of crush = (pounds/inch)/inch = lbs/in.². Points displaced by crush forces are constrained to move along the radial line from the vehicle center of gravity to the initial position of the displaced point before damage occurred. The program simulates crush by adjusting each of the radii in increments of $\Delta \rho$ until the pressures exerted balance to within an allowed error of λ .

Since for solution stability there must always be a value of ρ tested for which the pressures balance to within λ , and the change in pressure per increment $\Delta \rho$ is $K_{V}\Delta \rho$, it is necessary that $\lambda > K_{V}\Delta \rho$ for both $K_{V}1$ and $K_{V}2$.

The program, in seeking equilibrium, will increment ρ by $\Delta\rho$ up to 200 times. It this is insufficient, the message ISTOP = 7 will appear on the output, and execution will terminate. If the condition $\lambda > K_V | \Delta\rho$, $K_V 2\Delta\rho$ was satisfied, then a larger value of $\Delta\rho$ is needed (e.g. to .3 inches) when adjusting $\Delta\rho$ or K_V , be sure to adjust λ also if necessary.

14:1 CO Co Coefficients of assumed parabolic variation restitution. The car body is to have some the degree of restitution varying non-linear of deformation. Recommended values are .06: 10 ⁻³ , and 4.7381 x 10 ⁻⁵ , respectively. (To field of 8 columns the latter two should be 3.5417-3 and 4.7381-5, which are interprete notation).	elasticity with rly with the degree 423, 3.5417 x fit in the entered as
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(15:10) Two numerals 9999 appear in columns 77-80. Be sure all the numbered cards 1-14 have their number included, right justified, in the final integer field of each card.

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Brief Input Variable Description

	•	•		
Card No.	Program Variable	Analysis Variable	Definition	Units
1	T 0	-	Start time	Seconds
	TF	. •	End time	Seconds
· .	DTTRAJ	• •	Interval of integration at beginning and ending of run	Seconds
	DTCOLL	•	Interval of integration during collision contact	Seconds
	DTCOLT	- '	Interval of integration for 100 time increments subsequent to separation	Seconds
1.1.	DTPRN0		Output time interval	Seconds
11	CUVMIN		Vector velocity test for stop	Inches/Sec
	PSIDMN	-	'Angular velocity test for stop	Degrees/Saz
	IVEH0	-	Number of Simulated Vehicles (1.0 or 2.0)	•
2	XCP10	X'c10	Vehicle 1, initial X'c	Inches
	YCP10	y'c10	Vehicle 1, initial y'c	Inches
	PSI10	V 10	Vehicle 1, initial ψ	Degrees
• • •	PSI1D0	Ý 10	Vehicle 1, initial ψ	Degrees/Sec
. •.	U10	u ₁₀	Vehicle 1, initial U.	Inches/Sec
•	V10	V ₁₀	Vehicle 1, initial V	Inches/Sec
3	XCP20	, X' _{c20}	Vehicle 2, initial X'	Inches
•	YCP20	y c20	Vehicle 2, initial y'	Inches
	PSI20	¥ 20	Vehicle 2, initial # c	Degrees
•	PSI2D0	Ý 20	Vehicle 2, initial	Degrees/Sec
	U 20	U 20	Vehicle 2, initial U	Inches/Si-
	Y20	٧ ₂₀	Vehicle 2, initial Y	

21		
	~	7

Card No.	Program Variable	Analysis Variable	Definition	Units
4	A 1	a ₁	Vehicle 1, CG to F. Wheel	Inches
	Bl .*	Definition Units A 1 Vehicle 1, CG to F. Wheel Inches (2 (+) Definition Units Units Units Lb-Sec ² /In.		
	TRI	Τ,	Vehicle l. Average Tread	Inches
	FIZ 1	I 21	Vehicle 1, Yaw Inertia	Lb-S ec ² -In
	FMASS1	M_{1}	Vehicle 1, Total Mass	Lb-Sec ² 7In.
•	PSIR 10	ψ_{R1}	Vehicle 1, Rear Axle Angle (Damage)	Degrees
**	XF1	X _{F1}	Vehicle 1, CG to Front (+)	Inches
	XR1	$\mathbf{x}_{\mathbf{R}1}$	Vehicle 1, CG to Rear (-)	Inches
	YS1	Ysı	Vehicle 1, CG to Side (+)	Inches
5	AZ	a ₂	Vehicle 2, CG to F. Wheel	Inches
	B2	b ₂	Vehicle 2, CG to R. Wheel	Inches
	TR2	T_{2}	Vehicle 2, Average Tread	Inches
	FIZ2	I.Z2	Vehicle 2, Yaw Inertia	Lb-Sec ² /In
	FMASS2	M_{2}	Vehicle 2, Total Mass	· Lb-Sec ² /In
· .	PSIR20	V RZ	Vehicle 2, Rear Axle Angle (Daniage)	Degrees
- ,	XF2	X _{F2}	Vehicle 2, CG to Front (+)	Inches
	XR2	X _{R2}	Vehicle 2, CG to Rear (-)	Inches
	YS2	Y 52	Vehicle 2, CG to Side (+)	Inches
• .				
6	CSTF1(1)	C 11	Vehicle 1, RF Tire Cornering Stiffness	Pounds/Radian
•	CS TF1(2)	C 12	Vehicle 1, LF Tire Cornering Stiffness	Pounds/Radian
	CSTF1(3)	C 13	Vehicle 1, RR Tire Cornering Stiffness	Pounds/Radian
•	CSTF1(4)	C 14	Vehicle 1, LR Tire Cornering Stiffness	Pounds/Radian

Card	Program	Analysis	•	•
No.	Variable	Variable	Definition	Units
. 7	CSTF2(1)	C21	Vehicle 2, RF Tire Cornering Stiffness	Pounds/Radia
•	CSTF2(2)	C ₂₂	Vehicle 2, LF Tire Cornering Stiffness	Pounds/Radia
•	CSTF2(3)	C ₂₃	Vehicle 2, RR Tire Cornering Stiffness	Pounds/Racia
•	CSTF2(4)	C 24	Vehicle 2, LR Tire Cornering Stiffness	Pounds/Radia
				• ,
8	TBTQ1	-	Initial time for torque inputs, Vehicle 1	Seconds
	TETQ1	•	Final time for torque inputs, Vehicle 1	Seconds
	TINCOI	•	Time increment for torque inputs, Vehicle 1	Seconds
	NTBLQ1	. •	If # 0.0, do not read table	•

(1) Table of Traction (+) or Braking (-) Force at RF

Wheel, Vehicle 1 Card format 7F10.0, use three
to two hundred and one values for each wheel. The
number of entries for each wheel is computed as

TETQ1 - TBTQ1

TINCQ1 + 1.

Start the entries for each wheel on a new card. Seven entires per card.

- (2) Table of Traction (+) or Braking (-) Force at LF Wheel, Vehicle 1
- (3) Table of Traction (+) or Braking (-) Force at RR Wheel, Vehicle 1
- (4) Table of Traction (+) or Braking (-) Force at LR Wheel, Vehicle 1

		•		23 ,
Card No.	Program Variable	Analysis Variable	Description	Units
9	TBTQZ	•	Initial time for torque inputs, Vehicle 2	Seconds
	ŢETQ2	-	Final time for torque inputs, Vehicle 2	Seconds
•	TINCQZ	-	Time increment for torque inputs, Vehicle 2	Seconds '
,	NTBLQ2	•	If \(\psi \) 0.0, do not read table	•
•.	• • •	of Traction	(+) or Braking (-) Force	
•	• .	of Traction	(+) or Braking (-) Force	See comments
		of Traction Wheel, Ve	hicle 2	following card 8
-		of Traction Wheel, Ve	h (+) or Braking (-) Force	
10	TBPSF1	-	Initial time for steer inputs, Vehicle 1	Seconds
:	TEPSFI		Final time for steer inputs, Vehicle 1	Seconds
	TINCP1	-	Time increments for steer inputs, Vehicle 1	Seconds
	NTBLP1	•	If # 0.0, do not read table	-
	(1) Steer	Table (deg	rees) for RF Wheel, Vehicle 1	
•	(2) Steer	Table (deg	rees) for LF Wheel, Vehicle 1	
	(Se	e comment	s following card 8)	

	•	•	· · · · · · · · · · · · · · · · · · ·	
Card No.	Program Variable	Analysis Variable	Description	Units
11	TBPSF2	-	Initial time for steer inputs, Vehicle 2	Seconds
	TEPSF2	-	Final time for steer inputs, Vehicle 2	Seconds
	TINCP2	-	Time increments for steer inputs, Vehicle 2	Seconds
	NTBLP2	-	If # 0.0, do not read table	•
•			ees) for RF Wheel, Vehicle 2	•
	(2) Steer	Table (degr	ees) for LF Wheel, Vehicle 2	
	(Se	ee comment	s following card 8)	
•	¥ <u>*</u>		•	600
12	XBP(1)	X' _{B1}		Inches
• ;	YBP(1)	y' _{B1}	'	Inches
	XBP(2)	X B2	Points defining boundary between terrain zones	Inches
	YBP(2)	Y'B2	between terrain bonds	Inches
	XMU1	μ_1	Tire-Terrain Friction Coef- ficient at Zero Speed (Zone 1)	-
-	XMU2	μ_{z}	Tire-Terrain Friction Coef- ficient at Zero Speed (Zone 2)	-
	СМИ	Cµ	Coefficient of linear decrement of friction with tire speed	-
13	DELPS0	ΔΨ	Interval between radial vectors	Degrees
. •	DELRO0	Δρ	Increment of change in radius vector	Inches
	ALAMB	λ	Acceptable error in equilibrium	Lb/Inch
	ZETAV	ኢ 戈 _v	Minimum relative velocity for friction	Inches/Sec
	AKV(1)	K_{v1}	Load-deflection characteristic, Vehicle 1	Lb/In ²
•	AKV(2)	K_{v_2}	Load-deflection characteristic, Vehicle 2	Lb/In ²
	AMU	μ	Intervehicle friction coefficient	-

Card No.	Program Variable	Analysis Variable	- Description	Units
14	CO	C ₀	Coefficients of assumed parabolic	•
	Cl	c	Coefficients of assumed parabolic variation of coefficient of restitution	n -
	C2	c: I	with deflection	-

SECTION V

OUTPUT FORMAT

A. Discussion

The output of the SMAC program is largely self-explanatory, but to prevent any initial confusion it will be briefly discussed here. The output consists of printout and an optional graphic display.

The printout first gives a line-for-line transcription of the input record. This is followed by a table of input data in which the variable name and units, as well as the input value, are printed; thus when the user wants to find a particular variable it is not necessary to identify the specific input field. Directly under this table, control inputs (from torque and steering tables) are listed.

Next comes the main body of the printout. The first page, marked page 1 in the upper right hand corner, gives the following data for vehicle 1 in labeled columns:* Coordinate position of the center of mass, the heading angle, for forward and lateral velocities, the angular velocity, the acceleration in the forward and lateral directions, and the absolute magnitude of the total acceleration.

^{*} As always with the SMAC program, coordinates are in a Cartesian system with the positive x-axis at "North", positive y-axis at "East", and angles measured clockwise from the positive x-axis. For angular velocity, clockwise is positive, and for vector velocity forward and right are positive, backward and left negative. It should be noted that, although input are in inches, output are in feet.

Page 2 gives the velocity vector direction and tire tracks for vehicle 1. The velocity vector is the angle at which the car is moving with respect to its forward direction, measured clockwise from straight forward. Then the coordinate position of each tire is given for each time increment. The asterisk next to the coordinates denotes skidding tires.

Page 3 and 4 repeat this information for vehicle 2. The page after page 4, numbered page 1, begins again with vechicle 1 at the next time increment following vehicle 1's previous page 1. This 4-page pattern of vehicle 1's moving coordinate position and tire tracking (same for vehicle 2) repeats till the run ends.

At any time when the time increment changes, there is an announcement on the page prior to the first page including the new increment. The announcement includes values of the old and new increments, and the time of switchover. (For explanation of the roles of the respective Δt 's, see items 1:3-5 in the preceding section.)

At the end of this main table, there are miscellaneous subroutine messages and then a damage summary. The first part of the summary is a table of displaced points--any point moved by crush forces is given in two forms, polar and cartesian. Specifically, for each vehicle ther is a table of 4 columns. The first two give radius in inches and angle in degrees (measured clockwise from front center) of each displaced point with respect to the center of gravity of the vehicle. The other two columns give x and y coordinates of the displaced points in the vehicle-fixed coordinate system (origin at center of gravity, positive x-axis through front center, positive y-axis to right).

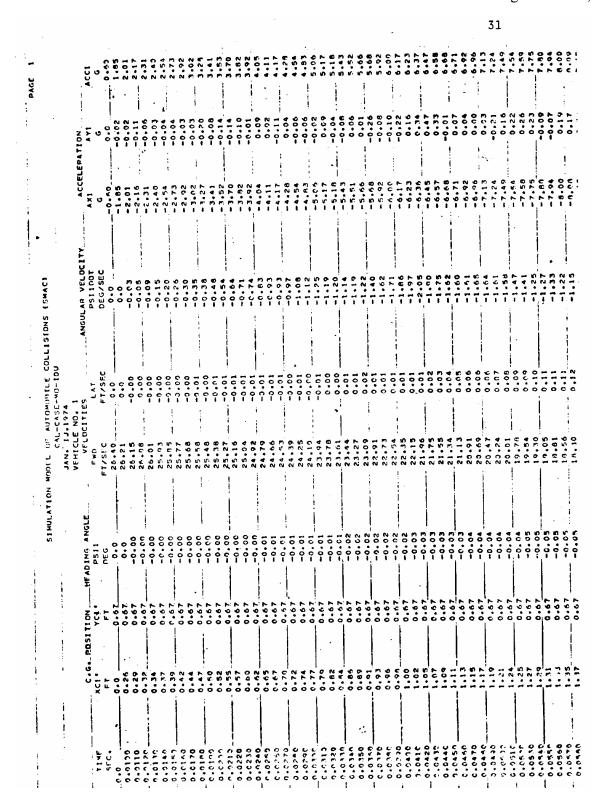
Below the table of displaced points for each vehicle is a concise discription of the damage, given in the following form for each vehicle: the beginning and end points of the damage area in polar coordinates, the angular coordinate of the midpoint of damage, the vehicle damage index (VDI), and ΔV . ΔV is the total change in velocity (the time integral of the absolute value of the acceleration) over the period where acceleration exceeds 1 g. given in mph.

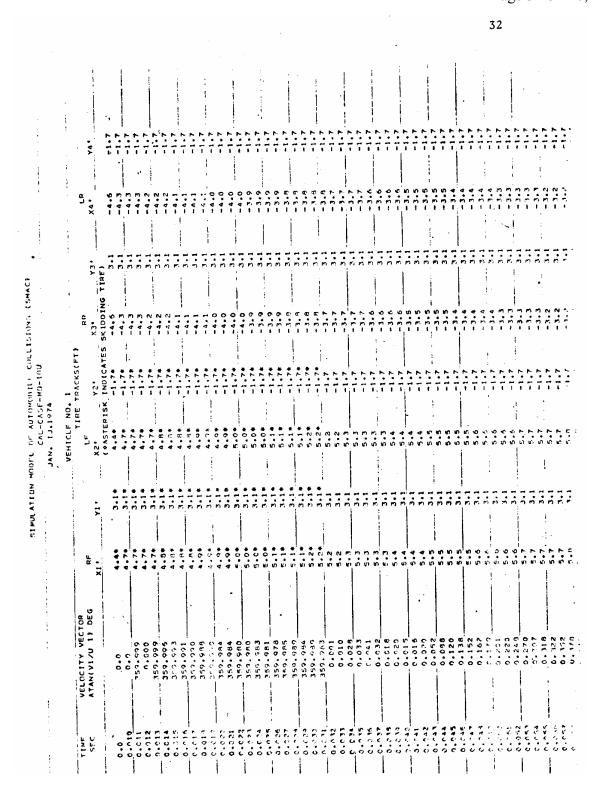
In addition to the printout, the SMAC program can create a graphic display of the simulated accident showing the position at impact, final position, damage, and tire tracks. The display consists of a heading, which includes the first two cards of input, the plot of collision, and a table of data. The plot includes labeled X' and Y' axes with a scale given at the bottom. The position of each car at impact is shown with dotted lines, and the final position with solid lines. Where damage has occurred, the solid line shows the damaged outline, with the original outline

superimposed with dotted lines. Vehicles are labeled vehicle 1 and vehicle 2, and heading is given by a triangle inside each car outline, pointing in the forward direction. A small circle indicates the center of gravity. Tire tracks are also shown—solid lines where the tire is skidding, dotted lines where it is rolling.

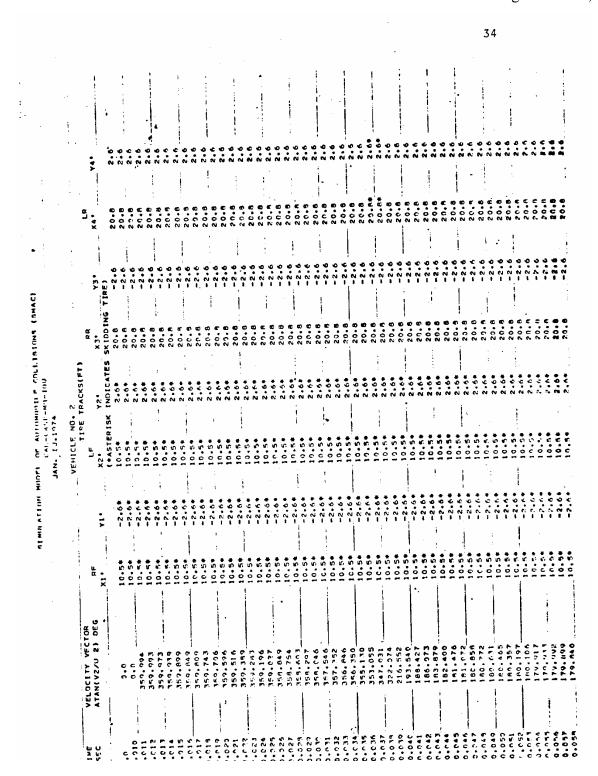
The table appearing beneath the plot gives the following information for vehicles #1 and #2: coordinate position of the center of gravity, heading angle, forward and lateral velocity in mph and angular velocity at impact, coordinate position of center of gravity, heading angle and remarks on presence or absence of motion at end of run, and figures for vehicle damage index and ΔV (See Section V(C), page 36 for graphic display).

data shown in IIIB)	CALCULATION_CONSTANTS	1	# 15.00 LV/IN		= 50,000 LB/(14*02)	N 1		# 0.47381E-C4 DEFLECTION			SHOOT NEEDEN SHOOT STRONG STRO	× 0.0 IN.	-N.2 O	0°.400	# 0.30000E-03	PROGRAM CONTPOL DATA		ONE OCO - I	0.050	- 0.001	LT # 0.010 ** INTEG.INIVINCEDS	E00.00	1 H	EHICLES = 2,	MORE DESCRIPTION OF THE PROPERTY OF THE PROPER	
LE (This output resulted from the input Simulation Model of AUTOMOBILE COLLISIONS (SMAC) CAL-CASE-WO-100		ENICLE NO. 2	XC20* = 187.000 NCHES	ATTES	A2 B 60.500 INCHES KVI	2 E 63.100 INCHES	12 H 38633, LIMBERGARZEIN CO	N20 = 0.0 OLGAEES .	# 100.500 #-119.600	452 = 39.630 INCHES	TIRE PROPERTIES	751 = -10253, LB/8AD XB1*	B -10250.	C(7) H 150195. ** XMUI	•	RHOBI TESTS	SION CRITERIA	PSILIMS a 10.000 DEGREES	. 000.061	* 350,000	01001	Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z Z	100154	IC ON	FROV.E	
B. Sample Computer Printout (INITIAL CONDITIONS	THICLE NO. 1	VC10*	DIMENSIONS AND INERTIAL		57.700 INCHES		0.0 DEGREES	# 85,700 1NCHES	YS1 # 35,700 INCHES		CORNERING STIFFNES	10250.	** ***********************************		PSIN NAMES TESTS	1	# 70,000 DEGREES				The second secon	•			





KC. G. POSI.T		i						
13.50 19.50 19.50 19.50			VEHICLE NO.	10. 2				
XC%.	¥	HEADING ANGLE	VELOCITIES	ທ	ANGULAR VELOCITY	ACCEU	ACCELERATION	ACC2
ម្តា ស្ត្រ	YC2.	9512	FKD	LAT	PS1200T	X 9		o
14.50	± 1	DEG) FO 6		0.0	-0.63	0.0	0.53
0 80 10 0 80 10 0 80 10	0.0	00.00		0.0	0.0	-1.53	-0.02	1.53
			2.68	00.0-	+0.0+	-1.65	70.04	60.
	00.0	160.00	2.62	00.0	-0.07	11.75	2000	
13.55	0.00	180.00	2.57	00.0-	11.01		10.01	1.93
15.54	00.0	189.00	2.51	00.0-	01.01	-2.02	-0.06	2.03
15.54	00.0	1 80.00	2.44	00.0	22.01	-2.16	-0.07	2.16
15.54	00.0	180.00	2.36	10.01	-0-75	-2.30	90.0-	2 • 30
15.54	00.0	180.00	200	10.0	86.00	-2.37	-0.07	2.37
15.54	00.0	20.001	2.18	10.0-	-0.41	-2.55	-0-19	2.55
		00.001	2.07	10.0-	-0.49	-2.65	-0-11	2.65
50.01	200	00.00	60.	20.0-	-0.53	-2.73	*0.16	2.74
15.53	00.0	00.001	00	20.0-	-0.59	-2.86	-0-17	2.85
F 5 5 7 7		00.001	0 2	-0.02	-0.63	-2.94	-0-	2.95
20.61			12.1	-0.02	-0.63	-3.02	90.0-	3.02
15.52		00.04		6.6	69.0-	-3.10	00.0	. CI.E
25.51	0.0	67.67		F.C. 6	-0.74	-3.15	50.0-	3.15
15.52		170.00	1.41	F0.0-	-0.74	-3.20	-0.1.	3.20
70.01		179.00	1.5	-0.03	44.0-	-3.28	0.0	3.5
	0000	179.99	1.20	+0.0-	-0.80	-3.45	11.0-	
18,81	0.0	179.99	1.09	₩0.0-	-0.A0	-3.67	11.0-	0
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15.51	0.00	170.99	0.72	40.0	D	60.0	41.0-	07.4
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				50.0-	-0.6B	-3.89	-0.03	3.80
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	000	179.98	-0.17	*0.0°	-0.52	-3.42	-0-11	ri e e en i
15.51	00.0	179.98	-0.28	₩0.0-	-0.54	44.6	0.12	e e
15.51	00.0	179.98	-0.39	₩0.0-	-0.57	55.55	22.0	
13.51	0.00	179.98	-0.50	50°0-	E . C .	VC. C.		
15.51	00.0	179.96	-0.62	n e e e	0 F			3.75
15.51	00.0	86.62		63.04	-0.34	3.78	10.0	3.78
18-81	9 6	10.07		10.0-	-0.33	-3.93	-0.03	3.93
	0.0	170.08	-1.10	-0.01	-0.43	-3.96	-0.06	3.96
15.52	00.0	179.98	-1.22	10.0-	94.01	E 0 . 4	0.0	
15.52	0.90	179.08	-1.38	10.0-	C		100	91.4
15.52	90.00	179.98	-1.48	10.0		96.44	0110	í n
15.52	0.00	179.98	24.			4.4	0.12	
15.52	00.0	179.98			0 1	-4.53	0.11	, 10. ◀
15.52	0.0	179.9R		00.0	E. C.	14.58	-0.12	.0.
15.53	00.0	00.01		00.0	02.0-	-4.68	-0-11	•
15.53	00.0	07.07.	ي (.	60.0	-0.70	-4.73	0.00	7.0
	2000			0.0	-0.7.	11.6-	10.0	
				14,0		* * * * * * * * * * * * * * * * * * * *	-	

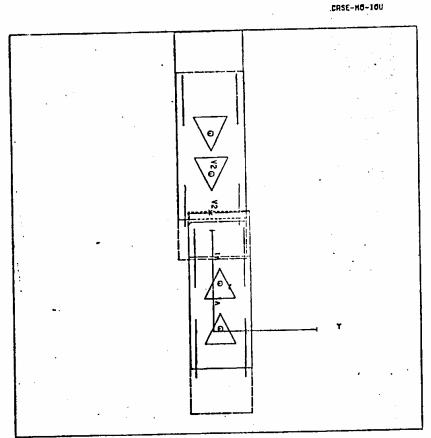


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	TATATE TO THE STAND THE ST

C. Sample Graphic Display(Derived from the input data of IIIB)

GRAPHIC DISPLAY OF OUTPUTS OF ACCIDENT RECONSTRUCTION

COLLISION AND TRAJECTORY



AXIS INTERVALS ARE 10. FEET

900	ACCOMMINATION POSITIONS AND VELOCITIES AT IMPACT DISPLAYED FINAL POSITIONS							AEHICLE .					
		E.S. 72		MERGING				E.G. P	BSITICH	HEADING		DANRGE	
•		EC.	YCI	PSIL	FHO	LATERAL	ANSULAR	XC1F	TOIF	PSILE	REMARKS	INDICES.	Δv
		n.	FT.	DES.	HFH	RPH	0EG/3EC	FT.	fT.	DEG.			RPH
AEDICT	e • i	0.0	0.7	0.0	17.9	0.0	0.0	4.7	0.7	-0.2	IN METIEM AT 2.0 SEC AFTER INITIAL CENTAGE		18.5
MEDIC.	E • 5	15.8	6.0	180.0	1.9	0.0	0.0	19.5	-0.0	179.6	VEHICLE RT REST	IZFOEHI	11.3

SECTION VI SUGGESTED VALUES OF PARAMETERS

In the interest of simplicity, the presented preliminary evaluation of the SMAC computer program has made use of "typical" parameters other than weights for the different categories of vehicle size rather than actual parameters for the specific vehicles. Vehicles representative of four different size categories were selected to provide a basis for "typical" parameters. The following vehicles were included in the different categories.

1.	Subcompact	3.	Intermediate
	Volkswagen Beetle Toyota 1200 Datsun 1200	•	Chevelle Torino Coronet Matador
	Vega Pinto Fiat 850		Skylark

2.	Compact		•	4.	Full Size
	Maverick			•	Chevrolet
	Camero	-			Galaxie
	Dart			•	Polara
	Hornet				Ambassador
				•	Monterey
				•	LeSabre
					New Yorker
• .:	•		• 4		Fleetwood
		•			Continental.

On the basis of available dimensional and shipping weight information, and with allowances made for both liquid weight and two passenger loading, the following "typical" parameters have been either directly derived or estimated from available measured values for similar vehicles.



TABLE 1

TYPICAL DIMENSIONAL AND INERTIAL PARAMETERS FOR 1971-72 AUTOMOBILES

	1	2	3	4	<i>'</i> -
Parameter	Subcompact	Compact	Intermediate	Full Size	Units
a ·	44.7	52.7	57.3	60.5	Inches
Ь	46.6	54.8	59.7	. 63.0	Inches
T	51.2	57.7	60.0	63.1	Inches
k ²	1963.	2635.	2998.	3588.	Inches 2
- M	5.71	8.51	9.86	12.42	Lb-Sec ² /
$\mathbf{x}_{\mathbf{F}}$	74.7	85.7	94.8	100.5	Inches
$\mathbf{x}_{\mathbf{R}}^{-}$	-83.5	-100.0	-110.8	-119.6	Inches
Ys	31.1	35.7	38.4	39.6	Inches

For the following vehicle parameters, representative values have been found but no refinement has yet been attempted for the different categories of vehicle size other than the load-deflection characteristic of the peripheral structure, $K_{\mathbf{V}}$.

REPRESENTATIVE VALUES OF VEHICLE PARAMETERS

Parameter	Value	Units
(CSTF) _{1.2}	-10250.	Pounds/Radian
(CSTF) _{3, 4}	-10195.	Pounds/Radian
Cμ.	3 X 10 ⁻⁴	Seconds/Inch
к _V	{Full Size = 50 } Subcompact = 30 }	Pounds/Inch ²
c _o	0.06423	-
C ₁	3.5417 X 10 ⁻³	•
C ₂	4.7381 X 10 ⁻⁵	- .
μ	0.550	•
ΔΨ	2.00	Degrees
۵۶	0.20	Inches
λ	15.0	Lb/In
S	5.0	In/Sec

SECTION VII

REFERENCES

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- (3) McHenry, R.R., Segal, P.J., Lynch, J.P., Henderson, P.M., "Mathematical Reconstruction of Highway Accidents", Calspan Report No. ZM-5096-V-1, January 1973, HS 800 801. Gives Several case examples.
- (4) McHenry, R.R., "Approximation of Impact Conditions via Computer Simulation", Proceeding International Accident Investigation Workshop, Brussels, Belgium, Pilot Study on Road Safety for the Committee on the Challenges of Modern Society, NATO, June, 1973, National Highway Traffic Safety Administrations, Washington, D.C. 20590. Provides a concise overview of SMAC.
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- (6) Jones, I.S., "The Application of the SMAC Accident Reconstruction Program to actual Highway Accidents". Proceedings of Eighteenth Conference of the American Association Automotive Medicine, Society of automotive Engineers, Inc., Toronto, Canada, 1974. Illustrates use of the START program to generate initial conditions for the SMAC program.